

AERO DYNAMIC AXIAL FLOW FAN DESIGN TO STUDY THE IMPROVEMENT IN EFFICIENCY

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Abstract: Much of the overall energy use is attributable to the fans that are employed to circulate air in commercial and industrial buildings. There is a pressing need for energy-efficient fan designs since a large portion of the city's future buildings have not been erected, and because power prices are rising and supplies are becoming more scarce. This study used a retrofit experimental design to compare two fan designs with respect to the total labour required to attain specified output values. An analysis is conducted to determine the relative merits of an aerodynamic axial flow fan and a belt-driven centrifugal fan, both of which provide identical output characteristics. This study's axial flow fan has an aerodynamically optimised design that significantly lowers the overall power input, connected load, and rated motor KW. The smaller air handling unit is an extra perk of this fan type. An analysis and documentation of the existing centrifugal fan design's actual performance characteristics are provided. A brand new, aerodynamically-designed axial flow fan has been built to meet these exacting standards. A thorough performance study is conducted once again after the retrofitting process to evaluate the optimised fan design's performance. The system's efficiency is computed both before and after the refit, taking into account the fan's mechanical efficiency. Using computational fluid dynamics (CFD), we were able to model the two designs' performance and examine how changing several design characteristics, such as the rotor blade twist, guiding vanes, and hub shape, affected the efficiency. Using the Ansys flow solver, we can run the simulation and find out how much power the fan needs to produce the airflow we want. The viability of replacing the existing fan with a new, more efficient one is evaluated by calculating the return on investment (ROI) of the new design using a ROI approach. It is now possible to cut

HVAC energy use by as much as 30 percent with this innovative, eco-friendly solution. Further optimisation of the design for improved efficiency and performance characteristics may be achieved with the use of computational fluid dynamic analysis of the system. Topics covered include aerodynamic axial flow, retrofitting, aerofoil guidance, and performance parameters for computational fluid dynamics.

Introduction

The circulation of air is fundamental to all living things. Fan is the mechanical device that was created to meet this need. In commercial and industrial air movement systems, fans account for over 40% of the total power consumption. Better energy-efficient fan designs might cut fan power consumption by as much as half. According to comprehensive research, the typical efficiency of industrial fans range from 33% to 55%. Fan inefficiency is mostly caused by the design of the fan's impeller or rotor. The overall mechanical efficiency of the fan has the potential to reach 92% with this optimisation. There is a pressing need for energy-efficient fans since a large portion of the city's new buildings are still under construction, and because power prices are rising and supplies are becoming more scarce. The rated motor kW, connected load, and total power input of the axial flow fan used in this research may be significantly decreased with the aid of its optimised aerodynamic design. The smaller air handling unit is an extra perk of this fan type. The amount of force needed to pump the specified air volume is dictated by the drag that is produced by the blade design. Reduced drag from an aerofoil's design means less thrust is required to pump large air volumes, which means less power consumption from the engine.

set up to turn the fan's impeller. In order to improve the fan's efficiency, we looked at a number of aerodynamic design characteristics. Industry standards are used to realistically quantify the enhanced performance. We will submit the revised design to industry experts for execution so that we may transition to renewable energy sources to power these machines, which will have a good effect

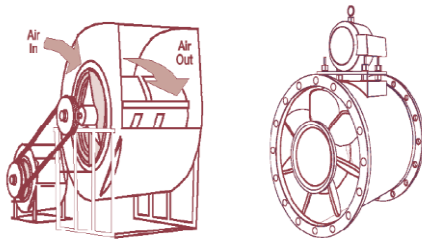


Figure 1.1. Types of Fans

a) Centrifugal Fan: Types

The major types of centrifugal fan are: radial, forward curved and backward curved. Radial fans are industrial workhorses because of their high static pressures (upto 1400mm WC) and ability to handle heavily contaminated airstreams. Because of their simple design, radial fans are well suited for high temperatures and medium blade tip speeds. Forward-curved fans are used in clean environments and operate at lower temperatures. They are well suited for low tip speed and high-airflow work - they are best suited for moving large volumes of air against relatively low pressures. Backward-inclined fans are more efficient than forward-curved fans. Backward-inclined fans reach their peak power consumption and then power demand drops off well within the reusable airflow range. Backward-inclined fans are

on the environment.

Types of Fans

Considerations like as efficiency, space constraints, material type, flow rate, pressure, and volume dictate the choice of fan and blower. Different kinds of fan designs have different efficiency. Centrifugal flow and axial flow are the two main types of fan motion. When air enters and exits a centrifugal flow device, the direction of the airflow may be either forward curved, backward curved, inclined, or radial. Axial flow (also known as propeller, tube, or vane axial) occurs when air flows into and out of a fan in a straight line.

known as "non-overloading" because changes in static pressure do not overload the motor.

b) Axial Flow Fan: Types

The major types of axial flow fans are: tube axial fan has a wheel inside a cylindrical housing, with close clearance between blade and housing to improve air flow efficiency. The wheel turns faster than propeller fans, enabling operation under high 65%. Vane axial fans are similar to tube axial, but with addition of guide vanes that improve efficiency by directing and straightening the flow. As a result, they have a higher static pressure with less dependence on the used generally for

and moderate temperatures. They experience a large change in air flow with small changes in static pressure. They handle large volumes of air at low pressure or free delivery. Propeller fans are

often used indoors as exhaust fans. Outdoor applications include air-cooled condensers and cooling towers. Efficiency is low approximately 50% or less.

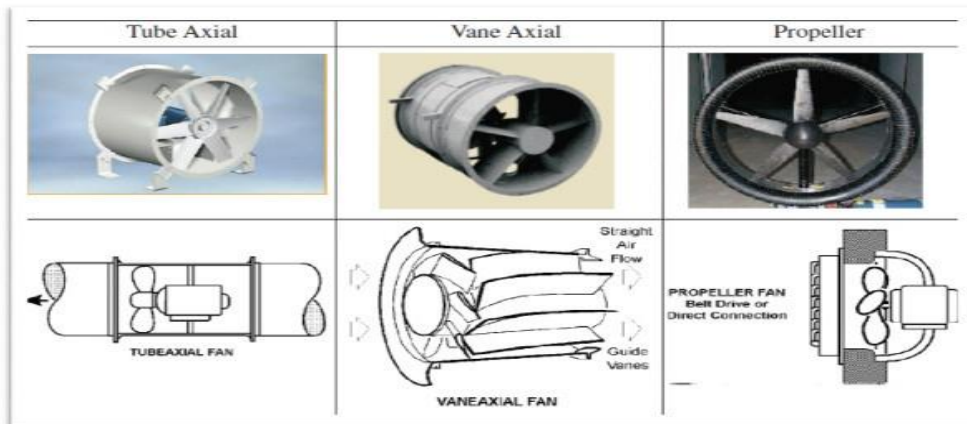


Figure 1.2. Types of Axial Flow Fans

1.2 Types of blade designs

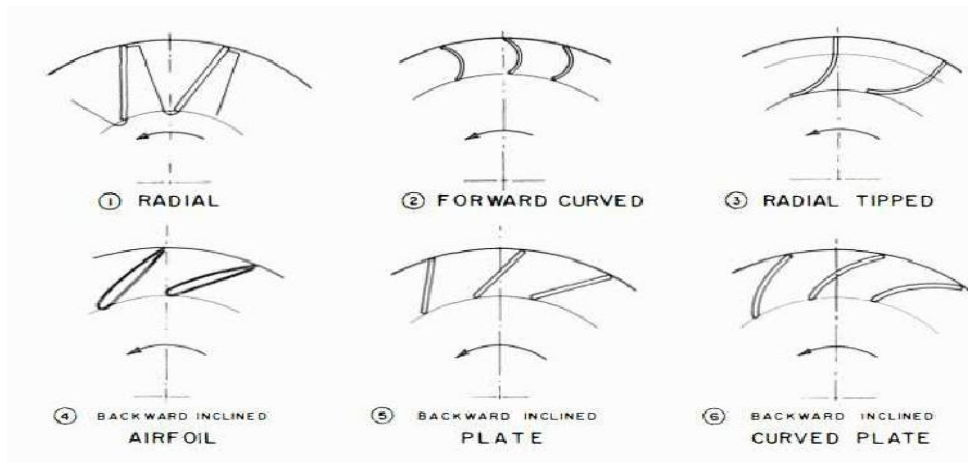


Figure 1.3. Types of Blade Designs

Radial blades: Forming a rotor which is essentially a large paddle wheel, this design results in a relatively inefficient fan with power consumption higher than that using the much more common backward inclined blade. Its inherent mechanical strength and resistance to wear and tear is generally used when high quantities of abrasive dust are present in the gas stream, or when very high

temperatures are expected.

Forward curved: A forward curved centrifugal fan is characterized by its cylindrical shape and lots of small blades on the circumference of the impeller. In the examples shown below, the fan rotates in a clockwise direction.

Backward curved
 An inefficient and strange shape, this design is a cost-effective alternative to the backward curved design but

with flat plate blades instead of curved. This results in a slightly lower efficiency, compensated by the ease of fitting of the liner

Table 1. Types Of Fans, Characteristics And Applications

| Centrifugal Fans | | | Axial Flow Fans | | |
|------------------------|---|---|-----------------|---|---|
| Type | Characteristics | Typical Applications | Type | Characteristics | Typical Applications |
| Radial | High Pressure, Medium Flow. Efficiency close to tube axial fans. Power increases continuously | Various Industrial applications, suitable for dust laden, moist air /gases | Propeller | Low Pressure, High flow, Low efficiency, Peak efficiency close to point of free air delivery (zero static pressure) | Air circulation Ventilation, exhaust. |
| Forward Curved Blades | Medium Pressure, high flow, dip in pressure curve, efficiency higher than radial fans, power rises continuously | Low pressure HVAC, packaged units suitable for clean and dust laden air/gases | Tube-axial | Medium Pressure, high flow, higher efficiency than propeller type, dip in pressure-flow curve before peak pressure point. | HVAC, drying ovens, exhaust systems |
| Backward curved blades | High pressure, high flow, high efficiency, power reduces as flow increase beyond point of highest efficiency | HVAC, various industrial applications, forced draft fans, etc. | Vane-axial | High pressure, medium flow, dip in pressure-flow curve, use of guide vanes improves efficiency exhausts. | High pressure applications including HVAC systems |
| Airfoil type | Same as backward type, highest efficiency | Same as backward curved but for clean air applications | | | |

Methodology of Design Study

The centrifugal fan, however, is more difficult, particularly when high efficiency is required. The blades are given an aerofoil shape to decrease the resistance created for the flow of air. Guide vanes are also made in aerofoil shape and the profile is given such that the leading edge of the aerofoil first comes into contact of the airflow to take the most of the benefit from the aerofoil design. Apart

from the aerofoil design the rotor blades are twisted along the length which is unique design optimization done in this fan design, the benefit of which is discussed in detail further.

2.1 Design criteria of a fan:-

The key factors of design criteria of a fan are , specifications of the fan, selection for the application, impeller stressing, critical speed of the rotor, field installation and fan sound.

Table 2. Specifications of Fan

| | |
|------------------------------------|-------|
| Volumetric flow, m ³ /s | 11.20 |
| Static Pressure, Pa | 640 |
| Density, Kg/m ³ | 1.20 |
| Speed, rpm | 1450 |
| Fan diameter, m | 0.9 |

2.2 Factors Aiding In Increased Efficiency Of Fan

The contact of the air with the first physical element of the fan itself starts the relation between the performance of the fan directly resulting the design criteria. The inlet cone has a very vast effect on the sound level of the fan. If a smooth transition with the help of inlet cone is not provided then the air cuts across the

diameter of the fan and high sound levels are experienced. Hence having an inlet cone helps in reducing fan sound levels. Similarly the hub blades and guide vanes along with the tip clearance between the blades and the rotor casing have influence on the fan performance.

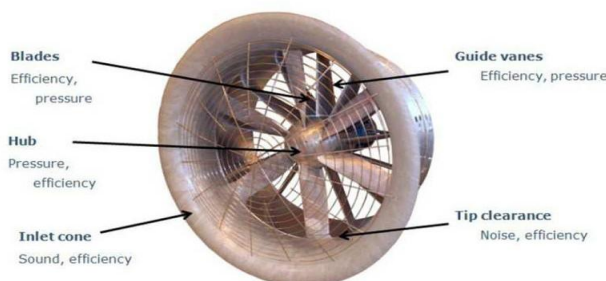


Figure 2.1 Aerodynamics of high efficient Axial flow fan

The term Axial flow fan indicates that the air flow through the fan is in an approximately axial direction. On the inlet side, as the flow approaches the fan blades, the direction of the flow is axial, in other words, parallel to the axis of rotation, provided there are no inlet vanes or other restrictions ahead of the fan wheel. The fan blade then deflects the air flow. A propeller is a mechanism designed to produce a reactive force or push, when submerged in a fluid medium. The propellers are aerodynamic elements that are composed of a hub or central core and a number of blades. The operating principle of axial-flow fans is simply deflection of air flow. Past the blade, therefore, the pattern of the deflected air flow is of helical shape, like a spiral staircase. This is true for all three types of axial-flow fans: propeller fans, tube axial fans, and vane axial fans.

2.3 The Aerofoil design of the blade

An aerofoil is a stream line shape. Its main application is as the cross section of a

airplane wing. Another application is as the cross section of a fan blade. There are symmetric and asymmetric aerofoils. The aerofoils used in fan blades are asymmetric. Fig.2.2 shows an asymmetric aerofoil that has been developed by the National Advisory Committee for aeronautics (NACA), it normally produces positive pressure on the lower surface of the aerofoil and negative pressure or suction on the upper surface. These suction pressures on the top surface are about twice as large as the positive pressures on the lower surface, but all these positive and negative pressures push and pull in approximately the same direction and reinforce each other. The combination of these positive and negative pressures results in a force F . This force F can be resolved into two components: a lift force L , perpendicular to the relative air velocity; and a drag force D , parallel to the relative air velocity. Fig.2.3 shows the forces acting on an aerofoil. The lift force L is the useful component ($R = W \sin \alpha$)

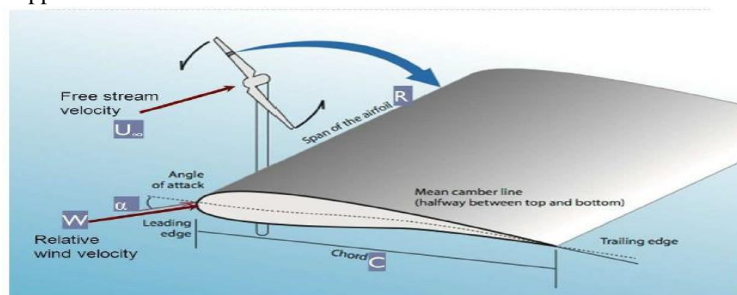


Figure 2.2. Shape of Asymmetric Aerofoil

In the figure 2.2 the leading edge is the point at the front of the aerofoil that has maximum curvature (minimum radius). The trailing edge is defined similarly as the point of maximum curvature at the rear of the aerofoil. The chord line is the straight line connecting leading and trailing edges. The chord length, or

simply chord, is the length of the chord line. That is the reference dimension of the aerofoil section. Camber line are the points halfway between chord and upper blade surface. Angle of attack is the angle between direction of airflow and the chord.

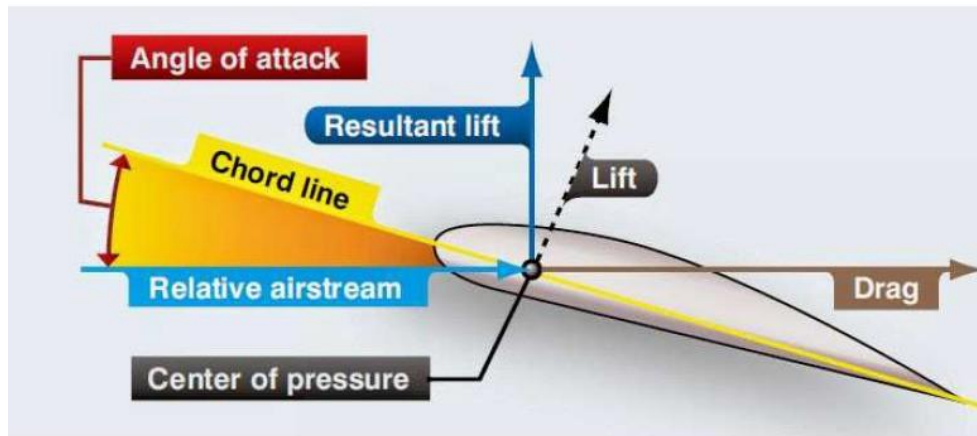


Figure 2.3. Forces Acting On Aerofoil

The lift that an aerofoil generates depends on the density of the air, the velocity of the airflow, the viscosity and compressibility of the air, the surface area of the aerofoil, the shape of the aerofoil, the angle of attack, and the angle of the aerofoil's angle of attack. However, dependence on the aerofoil's shape, the angle of attack, air viscosity and compressibility are where L is the lifting force, ρ is the density of air, v is the relative velocity of the airflow, s is the area of the aerofoil as viewed from an overhead perspective, and CL is the lift coefficient.

As with lift, the drag of an aerofoil depends on the density of the air, the velocity of the airflow, the viscosity and compressibility of the air, the surface area of the aerofoil, the shape of the aerofoil, and the

very complex. Thus, they are characterized by a single variable in the lift equation, called the lift coefficient. Due to the complexities of the lift coefficient, it is generally found via experimentation in a wind tunnel where the remaining variables can be controlled. Therefore, the lift equation is given by

$$\rho v^2 s CL$$

angle of attack. The complexities associated with drag and the aerofoil's shape, angle of attack, the air's viscosity, and air's compressibility are simplified in the drag equation by use of the drag coefficient. The drag coefficient is generally found through testing in a wind tunnel, where the drag can be measured, and the drag coefficient is calculated by rearranging the drag equation.

$$D = \left(\frac{1}{2}\right) \rho v^2 A C_D$$

In the drag equation, D is the drag force, ρ is the density of the air, v is the velocity of the air, A is a reference area, and C_D is the drag coefficient.

2.4 Twisted aerofoil blade

The blade experiences a relative wind velocity that is a vector sum of the actual windspeed (V) and the speed caused by the rotation of the rotor (Ωr where Ω is the rotational speed and r is the distance from the rotation axis). In order to produce lift, an aerofoil shape must be oriented so that its rounded leading edge is facing approximately into the airflow direction. But, the airflow direction for a wind turbine is actually a vector sum of the wind itself and the relative wind caused by the rotation of the blade through the air.

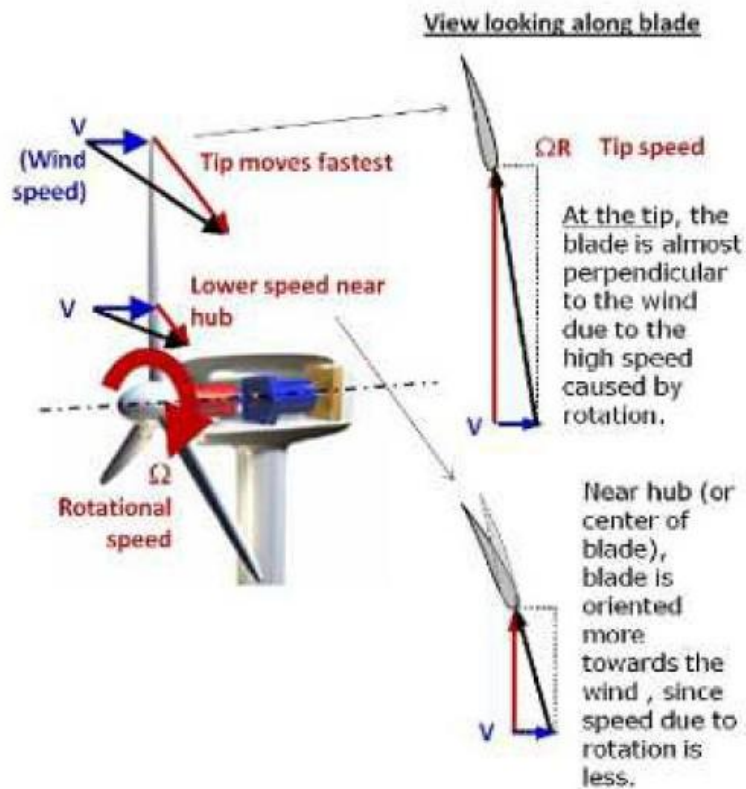


Figure 2.4. Mechanics of a twisted blade setup

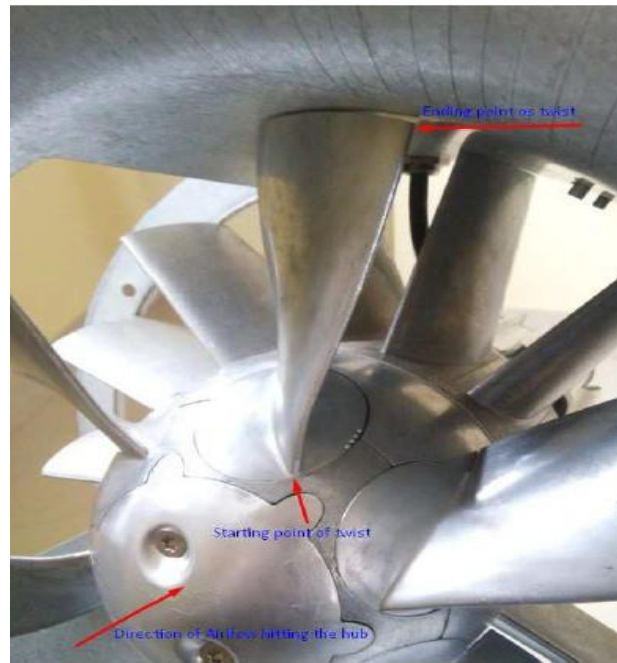


Figure 2.5. Mechanics of a twisted blade setup

Figure 2.4 shows the example of a wind mill blade. $TSR = \Omega R/V$, where Ω is the angular velocity of the rotor, R is the distance between the axis of rotation and the tip of the blade, and V is the wind speed. Since the speed of a rotating blade varies from the center to the tip, the angle with which the airflow encounters the aerofoil varies along the blade (see Figure 2.5). To account for this, the rotor blades must be twisted

It is important to note that the leading edge of the blade aerofoil mounted on the impeller is facing approximately into the airflow direction across the length of the blade. It is this design criteria which drastically help in reducing

the resistance to the movement of impeller.

2.5 Rounded Hub

A rounded hub benefit is shown in the figure above, usually axial flow fans are given flat hubs or not as much radial shape as needed, one such hub can be shown in the Figure 2.6. Before hitting the blades the air comes into contact with the hub, which experiences the fluid and transfers the flow to the blades, if the hub isn't correctly rounded as in our design it will have a loss in pressure and efficiencies of the fan at the surface of the blade near the hub doesn't have good contact to the laminar fluid hence disrupting the smooth flow of air reducing the ability of fan to create high pressure and efficiencies.



Figure 2.6. Rounded Hub and inlet cone



| Conventional Axial Flow Fan | Aerodynamically Designed Axial Flow Fan |
|---|--|
|  |  |

Figure 2.7. Comparison between conventional and our Aerodynamic fan design

2.6 The Guide vanes in fan design

The guide vanes at the outlet of air are provided to provide a near laminar effect to the fluid exiting the impeller, this will help to reduce the turbulence and also increases the efficiency of the fan as it helps to reduce the losses due to turbulence.



Figure 2.8. Asymmetric Aerofoil guide vanes of actual design

Turbulence not only causes increased noise levels but also causes massive pressure losses in the moving air. The design incorporates guiding vanes to make the system more efficient and to transfer the impeller's pressure without any loss. Figure 2.8 shows the asymmetrical form of the guiding vanes, which are aerofoils. The fixed guiding vanes' aerofoil design greatly enhanced the system's efficiency.

These vanes are used to transfer the impeller-generated pressure without any loss of pressure.

2.7 Adjusting the Guide Vanes' Profiles to Face the Leading Edge Into the Airflow

When air is released from the impeller, it meets the leading edge of the guiding vanes because the aerofoil's rounded leading edge is angled perfectly such that it faces towards the direction of the airflow,



as seen in the picture below.

Figure 2.9. In-line leading and trailing edge of blade and guide vane

The second optimised part is the small space between the blade tips, which results in less noise and improved efficiency. The rounded tip and inlet cone work together to improve efficiency by distributing airflow more uniformly. As

seen in the image below, the airflow is directed to the beginning point of the blade by the rounded hub without any loss, restriction, or diversion. The following are examples of standard axial flow fan designs for your reference.



Figure 2.10. Closeup view of blade tip clearance in our design

This study's aerodynamic fan design stands out due to the narrow space between the housing and the tip of the moving blade. Minimal blade tip

clearance reduces noise and improves efficiency. As shown up above, it's so thin that a business card might squeeze through.

3.1 Comparison between design of conventional axial fan and aerodynamic axial flow fan






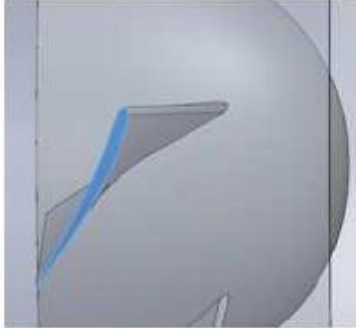

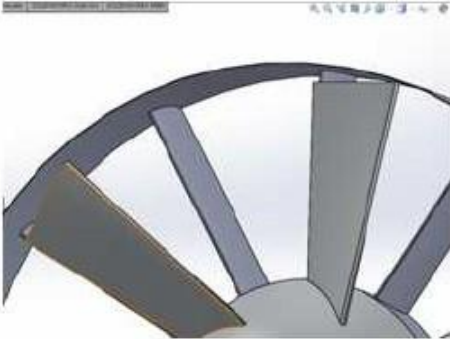


Figure 3.1. Cluster of fan designs showing conventional designs and an aerodynamic fan design

How an axial fan simulator works at its most basic level. The three primary parts of these procedures—computational domain, meshing, and solving—will serve as the framework for this discussion.

An entrance, a casing that contains the geometry of the blades, and an outlet make up the geometric domain. The produced mesh is very sensitive to the

actual dimensions and geometries of the three domain components. The fan's performance is affected by its most crucial aerodynamic component, the blade. Being the sole moving portion of the fan, the blade cuts through air and propels it forward. As a first step, we used reverse engineering to model the fan using all of the existing aerofoil drawings and the real design concepts.

Figure 3.2 below shows the 3D CAD models compared to the original design.

| Original Fan Model | 3D CAD Model for Analysis |
|--|--|
|  <p data-bbox="418 615 683 638">Aerofoil shape of the blade</p> |  |
|  <p data-bbox="345 1039 760 1060">Twist of the blade along the axis of rotation</p> |  |
|  <p data-bbox="430 1262 670 1283">A closer look at the twist</p> |  |


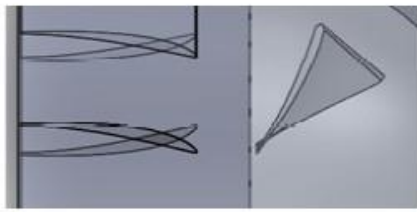
| Original Fan Model | 3D CAD Model for Analysis |
|---|--|
|  |  |

Figure 3.2. Table of pictures comparing original model and CAD model

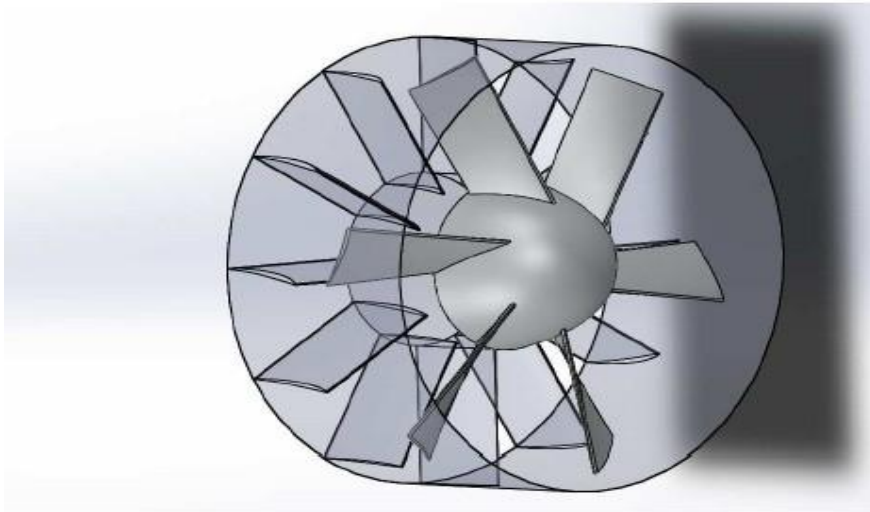


Figure 3.3. An isometric view of the Fan 3D CAD model for analysis

CONCLUSIONS:

An efficient fan design with an aerofoil-shaped impeller and guide vanes was investigated through experimental modelling, reverse engineering, and validation using the ANSYS CFX solver. The model was created in Solidworks. In comparison to a traditional centrifugal fan, the aerodynamically built fan significantly reduced power consumption, leading to a 68.75% savings and an increase in fan efficiency from 58.8% to 89.8%. Computer assisted design (CAD) and computer aided engineering (CAE) have helped model and simulate circumstances identical to those in the real experiment, and the findings obtained are within acceptable limits.

Experimental results showed that the new fan design had a number of advantages over the old centrifugal fan design, one of which was a much lower pressure need. Engineers will be able to choose the best fan model for future retrofit projects if this pressure analysis can be done in the simulation programme.

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